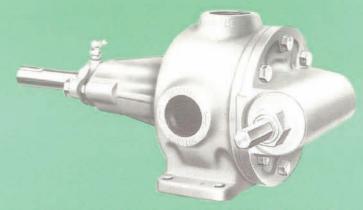
INSTRUCTION MANUAL WITH PICTORIALIZED PARTS LIST



SERIES 40

Positive Displacement Rotary Piston Pumps

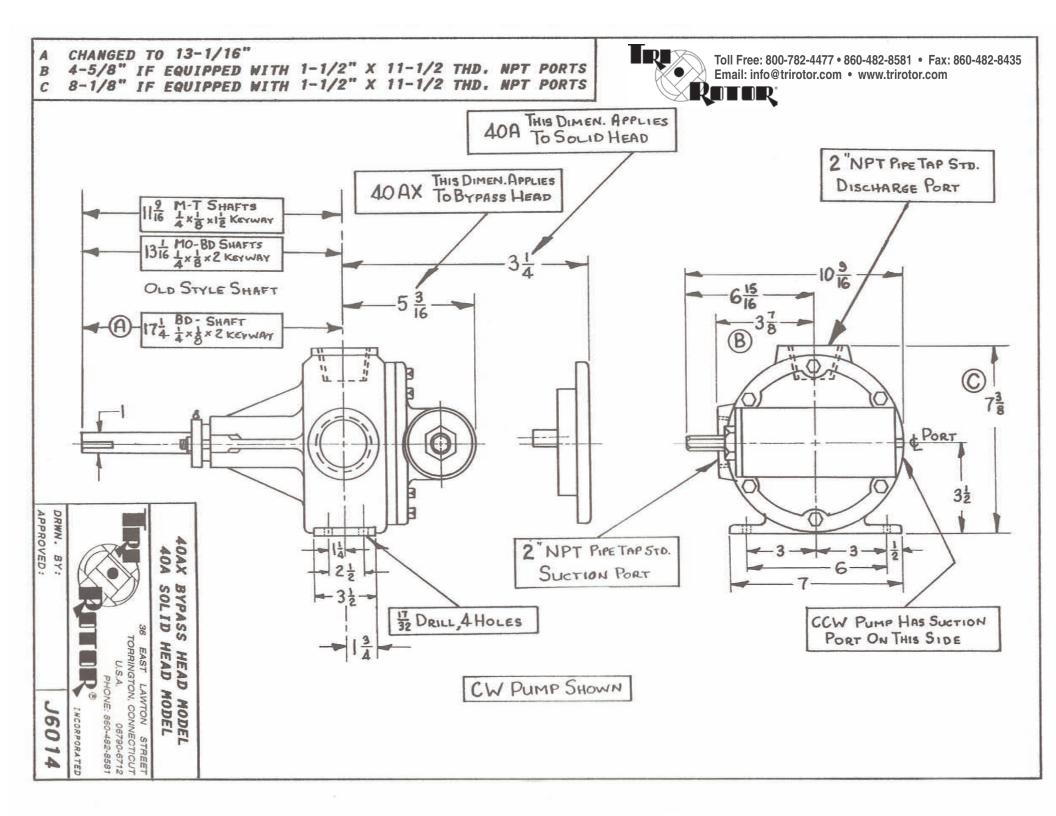


40 AX - BYPASS HEAD PUMP (PAGE 3)



40A - SOLID HEAD PUMP (PAGE 3)

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VISCOSITY - SPEED - GALLONAGE CHART 3	PICTORIALIZED PARTS LIST6, 7
SERIES 40 PERFORMANCE CHARTS3	TRI-ROTOR PUMPING PRINCIPLE Back Cover





SERIES 40 PUMPS

MODEL 40AX BYPASS HEAD PUMP, 40A SOLID HEAD PUMP RATED 40 GPM @ 540 RPM (MAXIMUM RATING 42 GPM @ 600 RPM)

PUTTING PUMP INTO SERVICE

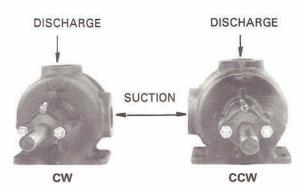
CAUTION: When receiving a pump, carefully check for damage, broken port seals, and misalignment incurred during shipping.

CORRECT PIPING HOOK-UP: The series 40 pump comes with two 2" N.P.T. ports (1 1/2" optional) designed for use with schedule 40 steel pipe. Connect piping based on direction of shaft rotation.

series 40 CW (CLOCKWISE) rotation pump has the suction port on the right side and discharge port on top (viewed with shaft end towards you).

SERIES 40 CCW (COUNTER CLOCKWISE) rotation pump has the suction port on the left side and discharge on top.

 Any pump may be run in reverse temporarily for such purposes as stripping lines etc.
 WARNING: CAVITATION COULD ENSUE



MOUNTING AND ALIGNMENT

The following will cause misalignment:

(1) Warped base plate (correct by shimming pump and drive components)

(2) Pipe strain (correct by using hangers or appropriate pipe supports)

If pump is connected to drive member by couplings, shim components until coupling halves are aligned.

To prevent misalignment of pump and drive components, fasten base securely in place using the foundation bolt holes provided.

- SHAFT SHOULD ALWAYS BE TURNABLE BY HAND
- As a last check before starting pump: remove gland nuts and slide packing gland out of housing. If gland does not slide back into housing without interference, pump is misaligned.

DIRECT MOTOR DRIVEN UNITS AND GEAR DRIVEN UNITS: Abutting shafts must be at least 1/8 of an inch apart and coupling inserts and / or chains should be loose enough to prevent end thrust on pump shaft.

BELT DRIVEN UNITS: An outboard bearing must be used to prevent side thrust on pump shaft. Pump shaft must be free to slide longitudinally through outboard bearing, so that rotor group will not be forced against pump case components. Align sheaves using straight edge or stretched cord.

OPEN GEAR DRIVEN UNITS: Proper alignment and engagement of gear and pinion can be checked by passing foil or cellophane through them. An outboard bearing should be used to prevent side thrust on pump shaft. Pump shaft must be free to slide longitudinally through outboard bearing, so that rotor group will not be forced against pump case components. CAUTION: Use gear and pinion set of same pressure angle such as furnished by factory, otherwise fibre motor pinion life will be short.

PACKING GLAND

The packing gland serves a dual function; first as packing follower and second as a bearing which, with the shaft housing bushing, forms a support for the rotor and shaft. As shipped from the factory the gland is LOOSE ENOUGH TO BE ROCKED BY HAND. At first start-up DO NOT tighten gland until pump has run long enough for packing to expand from absorption of pumpage. Thereafter, to adjust, tighten nuts evenly one quarter turn at a time and adjust enough to reduce leakage - NO MORE - a drop or two of the pumpage should normally drip from the gland every few minutes (except of course with mechanical seals, zero leak packing, or external scavenging systems). SHAFT SHOULD ALWAYS BE TURNABLE BY HAND. Keep shaft lubricated with appropriate lubrication through fitting provided.

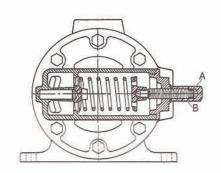
BYPASS HEAD MODEL 40 AX

This Tri-Rotor pump model has an integral dash pot relief valve in head. The standard spring can be set up to 50 PSI at which setting it will bypass full volume; the heavy duty spring can be set up to 100 PSI.

TO ADJUST BYPASS RELIEF PRESSURE

Remove hexagonal cap (A) and loosen locknut. Turn adjusting screw (B) in to increase pressure and out to decrease pressure.

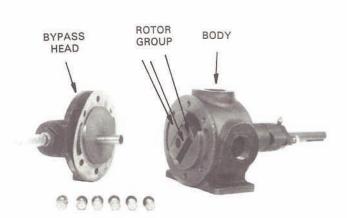
Tighten locknut and replace hexagonal cap, making sure copper gaskets are in place.



TO REVERSE DIRECTION OR ROTATION 40AX

The rotation of the **40AX** pump may not be reversed in the field for extended periods of time. The pump may be run in reverse **TEMPORARILY** for such purposes as stripping the lines etc.

 To reverse direction of rotation, a bypass head pump of opposite rotation must be ordered from the factory. See "putting pump into service" section on page 1.



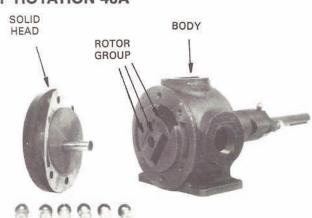
SOLID HEAD MODEL 40A

The model 40A has a solid head with the shuttle pin set in a fixed position to give constant volume for simple transfer service. A RELIEF VALVE SHOULD BE INSTALLED IN THE DISCHARGE LINE FOR PROTECTION.

TO REVERSE DIRECTION OF ROTATION 40A

The rotation of the **40AX** pump may not be reversed in the field for extended periods of time. The pump may be run in reverse TEMPORARILY for such purposes as stripping the lines etc.

 To reverse direction of rotation, a solid head pump of opposite rotation must be ordered from the factory. See "putting pump into service" section on page 1.





SERIES 40 PUMPS

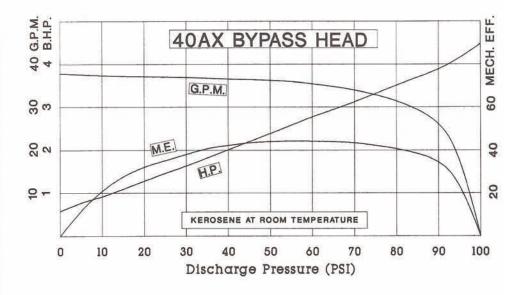
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PUMP SPEEDS FOR VARIOUS VISCOSITIES

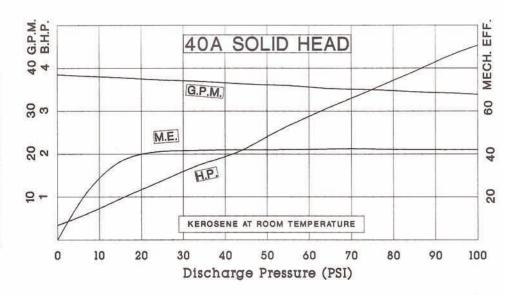
	SERIES 40			
Rating	40 GPM @ 540 RPM			
Displacement Factor	6.9 Gal./100 Revs.			
Port Size	1 1/2 "x11 1/2" THD NPT		D NPT	
SSU / CPS	RPM	GPM	Suct	
40 / 4	600	42	11/2	
100 / 20	600	42	11/2	
400 / 78	590	41	11/2	
600 / 125	580	40	11/2	
800 / 165	570	39	11/2	
1,000 / 200	560	38	11/2	
1,600 / 335	540	37	11/2	
2,000 / 410	530	36	2	
3,000 / 620	500	34	2	
5,000 / 1,060	480	33	2	
8,000 / 1,700	440	30	2	
9,000 / 1,900	420	29	2	



For Viscosities Below, Pump Must Have Relieved Rotor Group (For Sticky, Tacky Fluids.)

Port Size	2"x1	1 1/2" THI	NPT
10,000 / 2,150	500	34	21/2
15,000 / 3,100	480	31	3
20,000 / 4,250	440	28	3
30,000 / 6,500	380	25	3
40,000 / 8,610	320	21	3
50,000 / 10,800	280	15	3
75,000 / 16,210	210	11	3
100,000 / 21,625	150	8	3

GENERAL RULE: Viscous fluids which retain their "slipperiness" or which readily thin out with slight temperature increase or agitation do not require a relieved rotor group.



CAUTION: Suction piping diameter and length must be separately determined, regardless of pump port size, where (1) volatile liquids or (2) viscous pumpages are concerned. The sizes shown in the above chart are for suction lines no longer than 10 feet and containing no more than 2 pipe fittings.

TROUBLESHOOTING GUIDE

TROUBLE		TYPE OF PUMP		LOOK FOR	
INOUBLE	VARIABLE HEAD	BYPASS HEAD	SOLID	LOOK FOR	
		•	•	CW PUMP RUNNING CCW, OR VICE VERSA (1)* MOTOR WIRING REVERSED PIPING TO WRONG PORTS	
N		•	•	DISCHARGE HEAD TOO HIGH PIPING TOO SMALL, TOO LONG(3) VISCOSITY TOO HIGH (3)	
0				FLOW CONTROL TURNED DOWN TO ZERO	
F				PAWL PIN MISSING CONTROL SPRING - not adjusted - wound backwards - distorted - broken out of top or bottom spring plate	
L		•		RELIEF VALVE SPRING - not adjusted (3) - not in correct position (2) - spring broken	
O W				PLUNGER FROZEN IN BOTTOM POSITION - corroded parts - pumpage shear sensitive - dirt accumulation preventing movement	
VV		•		RELIEF VALVE - not fully seated (2) - stuck on valve guide pin - need lapping into seat - spring adjusting plate missing	
		•	•	INADEQUATE PRIMING CONDITIONS SUCH AS - suction line air leak - foot valve stuck - lift too great - altitude too high - vapor lock	
0.110747710				ROTOR GROUP WORN / MECHANICAL SEAL WORN OR BROKEN	
CAVITATING VIBRATING HIGH AMP, DRAIN		•		STARVED SUCTION LINE DUE TO - suction line restricted - viscosity too great for piping - RPM too high for viscosity (3)	
PUMP RUNNING		•	•	PACKING TOO TIGHT (1) MISALIGNED PUMP (1) INSUFFICIENT LUBRICATION OF SHAFT (1)	
HOT				TOO LONG RUNNING IN FULL BYPASS CYCLE OR ZERO STROKE	
		•	•	OVERSPEEDING (3) CAVITATION	
PUMP FROZEN CAN'T TURN SHAFT		•	•	PACKING TOO TIGHT (1) MISALIGNMENT (1) OBSTRUCTION IN ROTOR GROUP - rotor group part broken RUSTED PARTS- blush rust causing parts to sieze together	
		•	•	PUMPAGE - shear sensitive - congealed - caramelized - solidified TEMPERATURE OF ALL BRONZE OR BRONZE FITTED PUMP EXCEEDING 140° F ROTOR GROUP NOT RELIEVED	
NOISY		•	•	CAVITATION WORN ROTOR GROUP AIR LEAK INTO SUCTION LINE	
		•		PLUNGER OR VALVE BOUNCING DUE TO - suction line restriction - relief valve in discharge line reacting with pump spring setting - PIPING RESONANCE	
EXCESSIVE LEAKAGE FROM PACKING GLAND		•	•	PACKING NUTS INCORRECTLY ADJUSTED PACKING WORN MECHANICAL SHAFT SEAL WORN OR BROKEN SHAFT SCORED	
REDUCTION OF FLOW				PAWL PIN BROKEN CONTROL SPRING SETTING INCORRECT CONTROL PLUNGER STUCK	
OR PRESSURE		•		BYPASS SPRING SETTING INCORRECT (2) VALVE UNSEATED OR WORN (2)	
		•	•	PUMP WORN RESTRICTION OR TOO HIGH VISCOSITY IN SUCTION LINE (3)	
PREMATURE WEAR SHORT PUMP		•	•	MISALIGNMENT - end or side thrust on shaft (1) PACKING TOO TIGHT OR ADJUSTED INCORRECTLY DIRTY OR ABRASIVE PUMPAGE RUNNING PUMP DRY - repeated suction lift OVERSPEEDING (3) NON-LUBRICATING PUMPAGE OPERATING ABOVE 50 PSI & 350 RPM	
LIFE	():			FLOW CONTROL SET BELOW 25% CAPACITY FOR TOO LONG PERIODS SUCTION LINE RESTRICTED CAUSING PLUNGER "BOUNCE"	

^{*} NUMBERS IN PARENTHESIS PERTAIN TO PAGE NUMBERS WHERE MORE INFORMATION CAN BE FOUND.



MODEL 40AX (Exploded View)

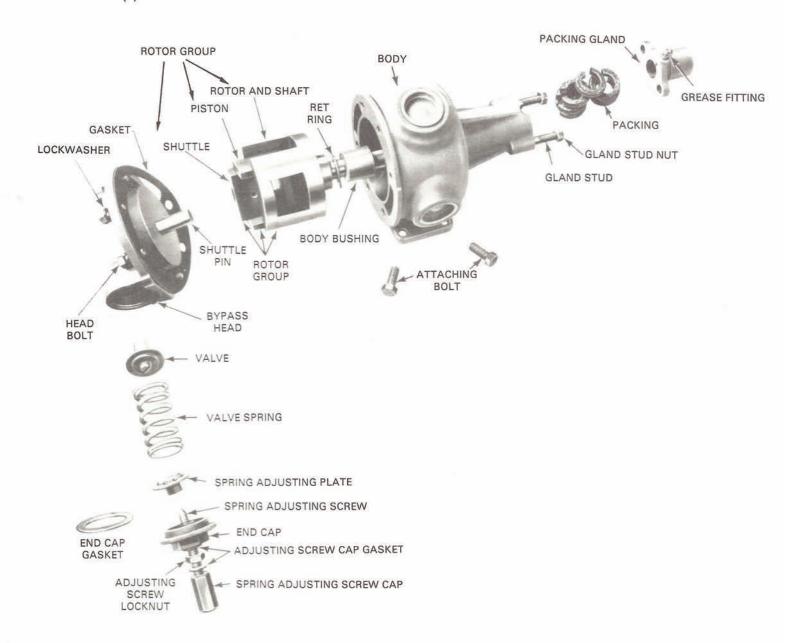
AVOID SYSTEM DOWN TIME!

KEEP ESSENTIAL SPARE PARTS ON HAND

- (1) ROTOR GROUP
- (2) PACKING GLAND
- (3) SET OF ALL GASKETS
- (4) PACKING
- (5) SHAFT HOUSING BUSHING

WHEN ORDERING PARTS ALWAYS GIVE:

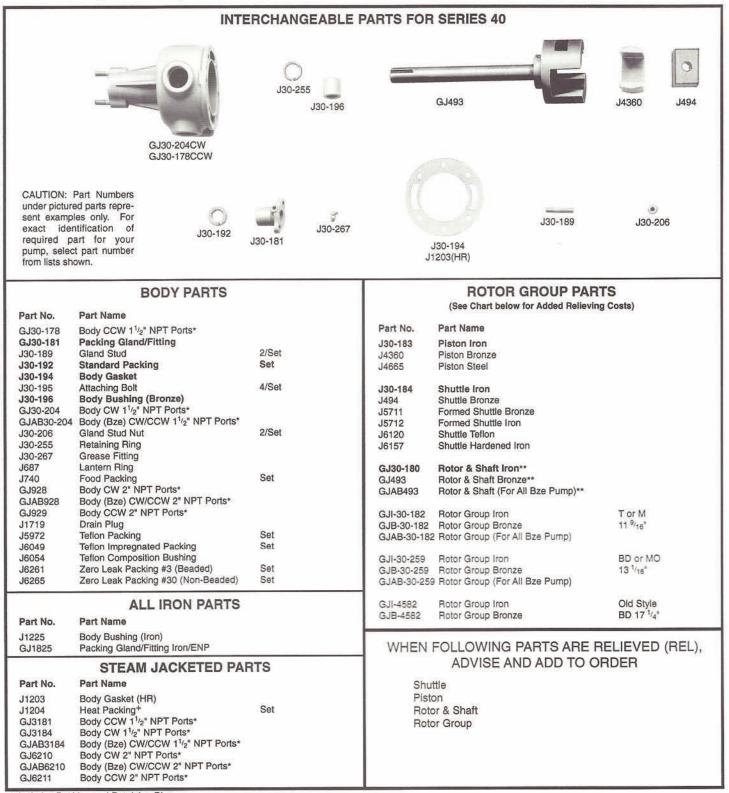
- (1) PUMP SERIAL NUMBER
- (2) ROTATION OF PUMP (CW or CCW)
- (3) SHAFT LENGTH (from port centerline)



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WHEN ORDERING PARTS ALWAYS GIVE:

- 1) PUMP SERIAL NUMBER
- 2) ROTATION OF PUMP (CW or CCW)
- 3) SHAFT LENGTH (from port centerline)



^{*} Includes Bushing and Retaining Ring.

NOTE: ALWAYS GIVE PUMP SERIAL NUMBER WHEN ORDERING PARTS.

ALL PRICES F.O.B. TORRINGTON, CT, SUBJECT TO CHANGE WITHOUT NOTICE.

^{**} Furnished with following shaft lengths: T or M 11 $^{9}/_{16}$ ", MO or BD 13 $^{1}/_{16}$ ", Old Style BD 17 $^{1}/_{4}$ ".

⁺ For temperatures under 500°F use J6049

WHEN ORDERING PARTS ALWAYS GIVE:

- 1) PUMP SERIAL NUMBER
- 2) ROTATION OF PUMP (CW or CCW)
- 3) SHAFT LENGTH (from port centerline)

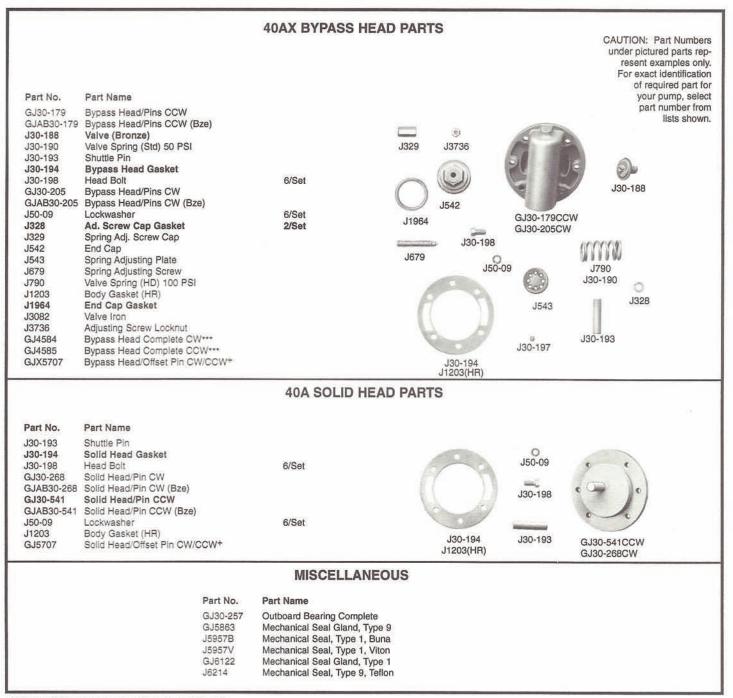
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40AX BYPASS HEAD PARTS AND 40A SOLID HEAD PARTS



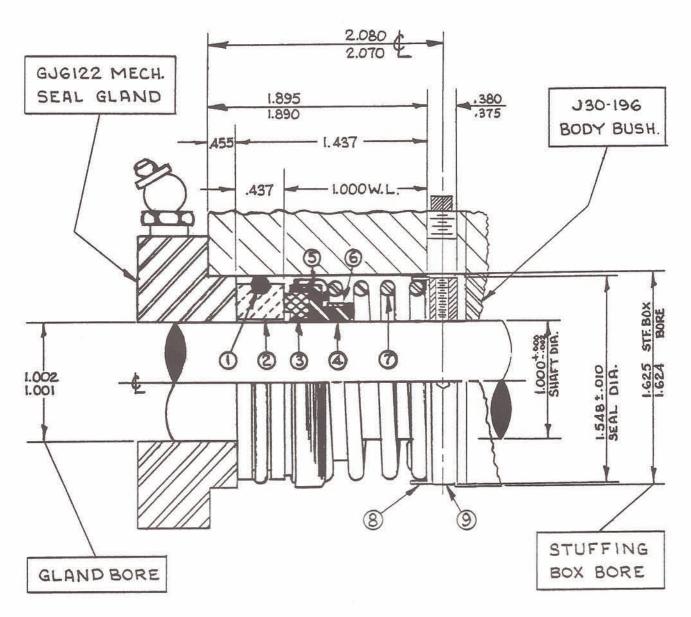
^{***} Advise if Standard or Heavy Duty Spring required

+ Specify Offset Pin position

NOTE: ALWAYS GIVE PUMP SERIAL NUMBER WHEN ORDERING PARTS.

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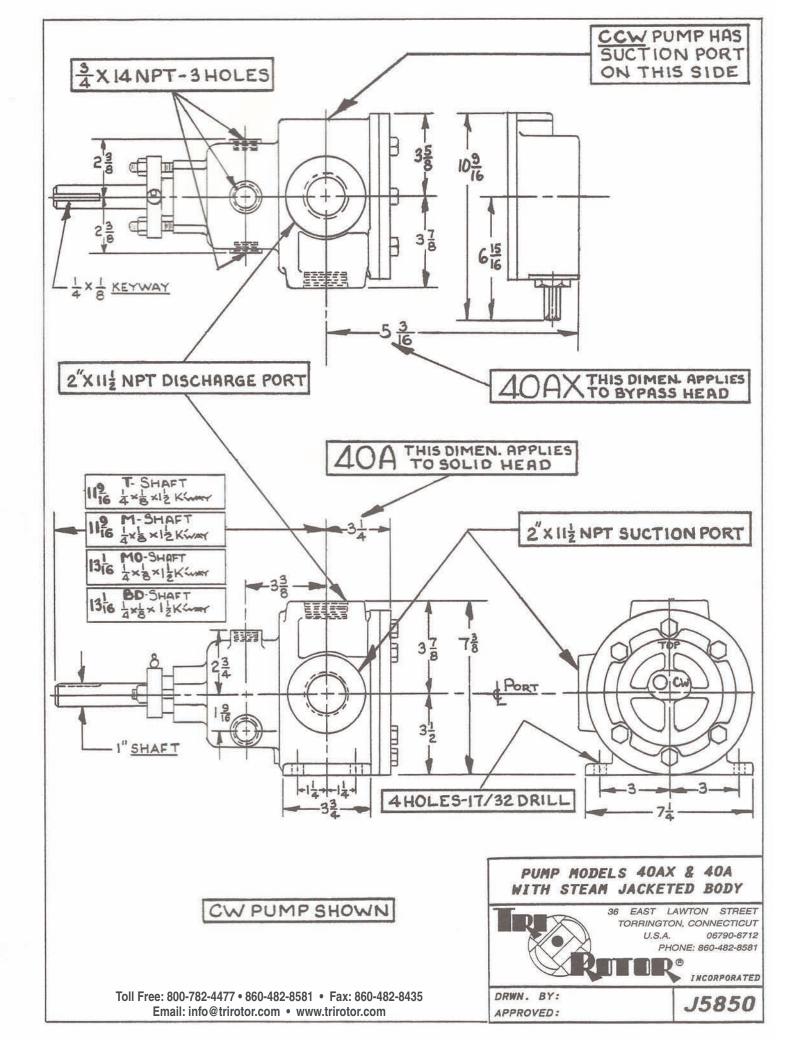
ITEM	NAME	QTY	MATERIAL
1	O-RING	1	VITONA
2	SEALSEAT	1	NIRESIST
3	SEAL FACE	ı	CARBON
4	FRICTION RING	1	VITONA
5	SHELL	I	STAINLESS STL.
6	BAND	1	STAINLESS STL.
7	SPRING	1	STAINLESS STL.
8	SPRING HOLDER	1	STAINLESS STL.
9	STOP COLLAR	1	PLATED STEEL

MODEL 40 TYPE I MECHANICAL SHAFT SEAL ASSEMBLY

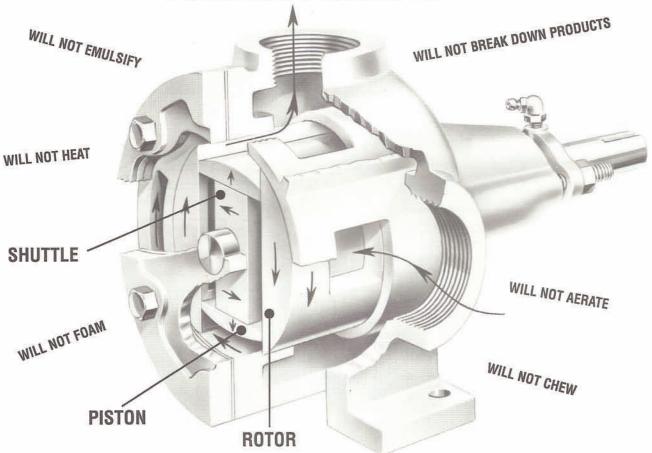


DRWN. BY:
APPROVED:

GJ5957







The mechanical principle of the Tri-Rotor Pump is explained as follows and incorporates the pump casing, the rotor, the piston, and the shuttle. The rotor is a liquid-tight fit within the casing, with the piston and shuttle being equally liquid-tight in their fit to each other and to the rotor. In operation the piston slides back and forth in the rotor slot and discharging from the opposite end. At the same time the shuttle slides back and forth within the piston slot (picture), drawing liquid through one rotor port and discharging through the other. The rotor, which functions as a rotating valve, channels the liquid from the intake port around through the casing and out the discharge port.

This action, while rotary, actually accomplishes the same type of pumping principle as a direct-acting piston pump. There are, therefore, two direct-acting pistons pumping through two cylinders, being valved by the rotary action of the rotor.

The reciprocating piston action is accomplished by the center bearing of the shuttle which rotates on a shuttle pin eccentric to the rotor shaft. Since the rotor is concentric with the shaft and the shuttle bearing is eccentric to the shaft, a reciprocating action of the piston and shuttle within their respective cylinder slots is created by revolving the rotor. Four overlapping strokes of the piston and shuttle for each revolution of the rotor create a smooth discharge with a pulsation reduced to a minimum. An extremely high volumetric efficiency is obtained because of the piston-type action and the liquid-tight fit of the moving members.

Highly viscous materials are easily handled with exceptionally high volumetric efficiency while thin, volatile materials are handled with little loss in slippage through the pumping members. Materials critical to agitation are handled with little or no mechanical beating, since the product is carried through the pump by piston action without being subjected to the combination centrifugal and gear or paddle agitation.

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